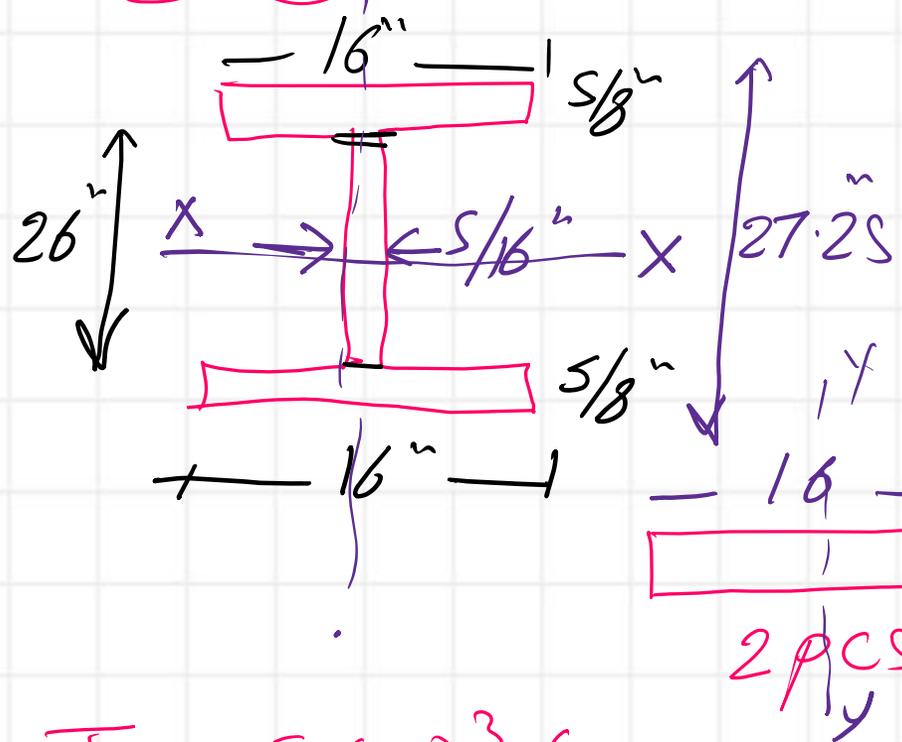


## Summary of Past-27 Content - LTR

- a) Find  $L_p$  the limiting effective unbraced length
- b) Find  $L_r$ : limiting effective unbraced length for Elastic LTB
- c) Find  $C_B$ , the coefficient of bending for beam AB  $\rightarrow$  4S' which is laterally supported at third points

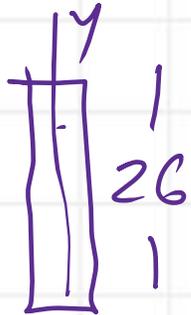
L.T.B



$$L_p = 300 r_y \sqrt{F_y}$$

We need to estimate  $I_y, A$

$$\frac{I_y}{A} = r_y^2 \Rightarrow r_y = \sqrt{\frac{I_y}{A}}$$



2 pcs  
y

$$I_y = \frac{5}{8} (16)^3 (2) + 26 \left(\frac{5}{16}\right)^3 \left(\frac{1}{12}\right) \frac{5}{16}$$
$$= 426.67 \approx 427 \text{ inch}^4$$

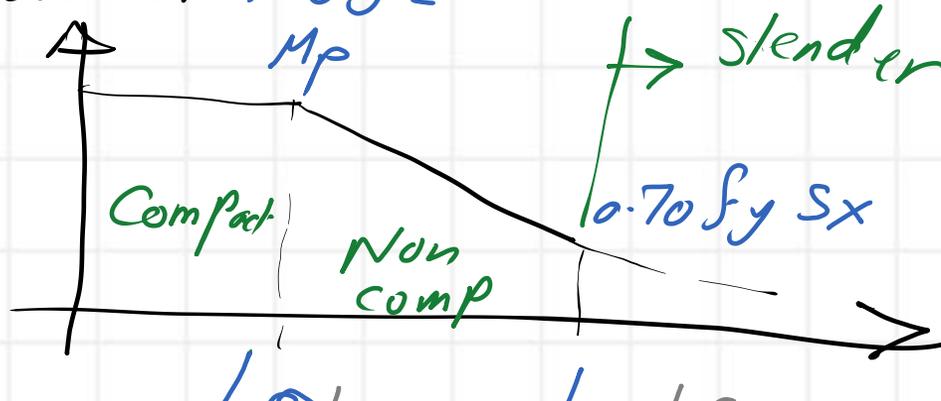
$$A = 2(16) \left(\frac{5}{8}\right) + 26 \left(\frac{5}{16}\right)$$
$$= 28.125 \text{ inch}^2$$

$$r_y = \sqrt{\frac{427}{28.125}} = 3.896 \text{ inch}$$

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$E = 29000 \text{ ksi}$ ,  $F_y = 65 \text{ ksi}$ ,  $L_b = 15'$

$C_b = 1$



$M_p = 1728 \text{ Ft-kips}$  **LTB**

$M_r = 1114 \text{ Ft-kips}$

$M_p = Z_x F_y = 319.06 (65) = 1728.24 \text{ FT.kip}$   
 $M_r = 0.7 F_y S_x = 0.7 (65) \left( \frac{293.8^2}{12} \right) = 1113.99 \text{ FT.kip}$

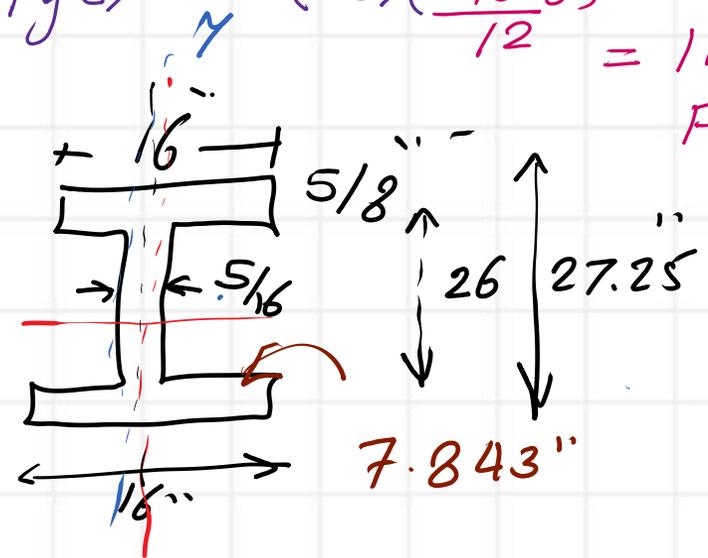
$\frac{L_p}{r_y} = \frac{300}{\sqrt{F_y}} = \frac{300}{\sqrt{65}} = 37.21$   
 $F_y = 65 \text{ ksi}$

$L_p = \frac{300 r_y}{\sqrt{F_y}} = \frac{300 (3.896)}{\sqrt{65}} = 144.93'' \Rightarrow 12'$

$L_p = 12.078'$

$L_b = 15'$  given

$L_b > L_p$



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# Beer chapter - 3

The determination of the stresses in noncircular members subjected to a torsional loading is beyond the scope of this text. However, results obtained from the mathematical theory of elasticity for straight bars with a *uniform rectangular cross section* are given here for our use.<sup>†</sup> Denoting by  $L$  the length of the bar, by  $a$  and  $b$ , respectively, the wider and narrower side of its cross section, and by  $T$  the magnitude of the torque applied to the bar (Fig. 3.44), the maximum shearing stress occurs along the center line of the *wider face* and is equal to

$$\tau_{\max} = \frac{T}{c_1 a b^2} \quad (3.40)$$

The angle of twist can be expressed as

$$\phi = \frac{TL}{c_2 a b^3 G} \quad (3.41)$$

Coefficients  $c_1$  and  $c_2$  depend only upon the ratio  $a/b$  and are given in Table 3.1 for a number of values of that ratio. Note that Eqs. (3.40) and (3.41) are valid only within the elastic range.

Table 3.1 shows that for  $a/b \geq 5$ , the coefficients  $c_1$  and  $c_2$  are equal. It may be shown that for such values of  $a/b$ , we have

$$c_1 = c_2 = \frac{1}{3}(1 - 0.630b/a) \quad (\text{for } a/b \geq 5 \text{ only}) \quad (3.42)$$

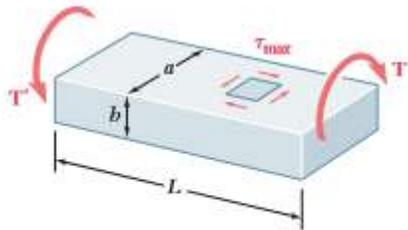


Fig. 3.44 Shaft with rectangular cross section, showing the location of maximum shearing stress.

Table 3.1. Coefficients for Rectangular Bars in Torsion

a/b	$c_1$	$c_2$
1.0	0.208	0.1406
1.2	0.219	0.1661
1.5	0.231	0.1958
2.0	0.246	0.229
2.5	0.258	0.249
3.0	0.267	0.263
4.0	0.282	0.281
5.0	0.291	0.291
10.0	0.312	0.312
$\infty$	0.333	0.333

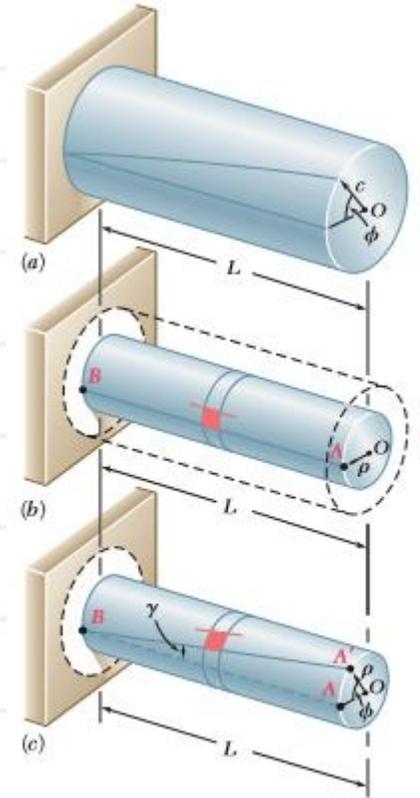
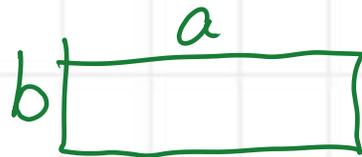


Fig. 3.13 Shearing strain deformation. (a) The angle of twist  $\phi$ . (b) Undeformed portion of shaft of radius  $\rho$ . (c) Deformed portion of shaft; angle of twist  $\phi$  and shearing strain  $\gamma$  share the same arc length  $AA'$ .

$$\phi = \frac{TL}{J_p G}$$



$$I_{\max} = \frac{T}{\frac{1}{3} a b^2}$$

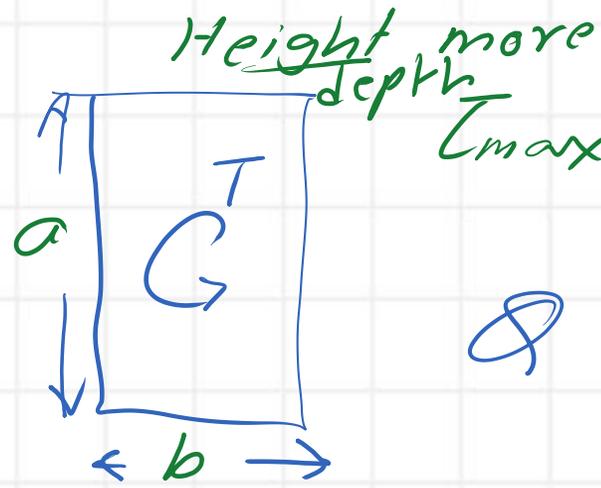
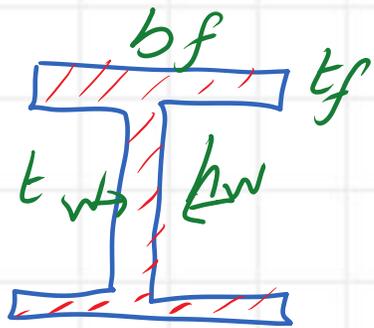
$$\phi = \frac{TL}{G (\frac{1}{3} a b^3)} \rightarrow J$$

$$f = \frac{M y_{\max}}{I}$$

$$I_{\max} = \frac{T \cdot R}{J_p G}$$

$$y_{\max} = \frac{R \phi}{L} = \frac{I_{\max}}{G} = \frac{T R}{J_p G}$$

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Height more than 5  
depth

$$I_{max} = \frac{TL}{C_1 ab^2}$$

$$\phi = \frac{TL}{C_2 ab^3 G}$$

$$a > 5b \quad C_1 = C_2 = \frac{1}{3}$$

$$I_{max} = \frac{TL}{\frac{1}{3} ab^2}$$

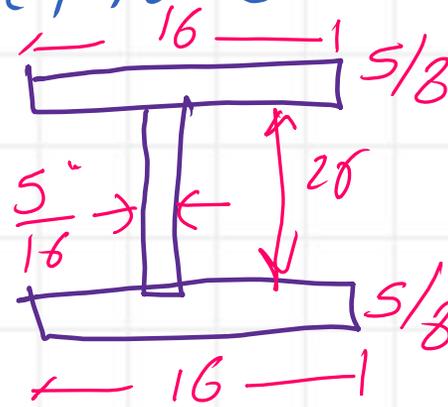
$$\phi = \frac{TL}{\frac{1}{3} ab^3 G}$$

J: Torsional Coefficient  $\rightarrow$  AISC  $= \frac{1}{3} ab^3$

$J = \sum \frac{1}{3} ab^3$  Case of I beam Rolled or T sections  
built up sections

$$L_r = 1.95 r_{ts} \frac{E}{0.7 F_y} \sqrt{\frac{J_c}{S_x h_o}} \sqrt{1 + \sqrt{1 + 6.76 \left( \frac{0.7 F_y S_x h_o}{E J c} \right)^2}}$$

27.25



$$a_1 = 16''$$

$$b_1 = \frac{5}{8}$$

$$a_2 = 26''$$

$$b_2 = \frac{5}{16}$$

$$J = \sum \frac{1}{3} ab^3 \Rightarrow \frac{1}{3} (16) \left( \frac{5}{8} \right)^3 (2) + \frac{1}{3} (26) \left( \frac{5}{16} \right)^3$$

$$= 2.8686 \text{ inch}^3$$

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$$S_x = 293.78 \text{ inch}^3 \Rightarrow 294 \text{ inch}^3$$

where

$r_y$  = radius of gyration about y-axis, in. (mm)

$r_{ts}$  = effective radius of gyration, in (provided in AISC Table 1

$J$  = torsional constant, in<sup>4</sup> (AISC Table 1-1)

$c$  = 1.0 for doubly symmetric I-shapes

$h_o$  = distance between flange centroids, in (AISC Table 1-1)

$$r_{ts}^2 = \frac{\sqrt{I_y C_w}}{S_x}$$

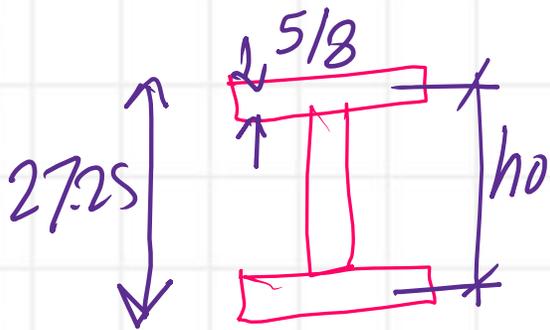
$$I_y = 427 \text{ inch}^4$$

$$C_w = \frac{I_y h^2}{4 c^2}$$

$$r_{ts} = \sqrt{\frac{\sqrt{I_y C_w}}{S_x}}$$

$$C_w = \frac{I_y h^2}{4 c^2}$$

$$c = \begin{cases} 1 & \text{for doubly-symmetric I-shapes} \\ \frac{h}{2} \sqrt{\frac{I_y}{C_w}} & \text{for channels} \end{cases}$$



$$h_o = h - t_f = 27.25 - \frac{5}{8} = 26.625 \text{ in}$$

$$c = 1$$

$$C_w = \frac{427 (26.625)^2}{4} = 75674.07 \text{ inch}^6$$

$$\frac{F}{0.7 F_y} = \frac{29000}{0.7(65)} = 637.36$$

$$r_{ts} = \sqrt{\frac{\sqrt{427(75674.07)}}{294}} = 4.397 \approx 4.40 \text{ in}$$

$$L_r = 1.95 r_{ts} \frac{E}{0.7F_y} \sqrt{\frac{Jc}{S_x h_o}} \sqrt{1 + \sqrt{1 + 6.76 \left( \frac{0.7F_y S_x h_o}{E J c} \right)^2}}$$

$$L_r = 1.95 (4.395) \left( \frac{29 \times 10^3}{0.7(65)} \right) \sqrt{\frac{J(1)}{294 (26.625)}} \sqrt{1 + \sqrt{1 + 6.76 \left( \frac{0.7(65) \cdot \frac{294}{29000} (26.625)}{J} \right)^2}}$$

$$= 61.7392 \sqrt{J} \sqrt{1 + \sqrt{1 + 6.76 \left( \frac{0.7(65) \cdot \frac{294}{29000} (26.625)}{J} \right)^2}}$$

$$S_x \approx 294 \text{ inch}^3$$

$$C_w = I_y h^2 / 4c^2$$

where

$r_y$  = radius of gyration about y-axis, in. (mm)

$r_{ts}$  = effective radius of gyration, in (provided in AISC Table 1)

$J$  = torsional constant, in<sup>4</sup> (AISC Table 1-1)

$c$  = 1.0 for doubly symmetric I-shapes

$h_o$  = distance between flange centroids, in (AISC Table 1-1)

$$r_{ts}^2 = \frac{\sqrt{I_y C_w}}{S_x}$$

$$C_w = \frac{I_y h^2}{4c^2}$$

$$c = \begin{cases} 1 & \text{for doubly-symmetric I-shapes} \\ \frac{h}{2} \sqrt{\frac{I_y}{C_w}} & \text{for channels} \end{cases}$$

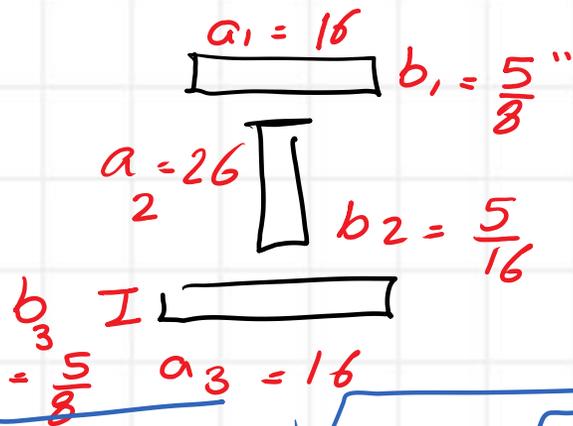
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$$L_r = 1.95(4.395) \left( \frac{29 \times 10^3}{0.7(65)} \right) \frac{\sqrt{J(1)}}{294(26.625)} \sqrt{\sqrt{\quad}}$$

$$= 61.7392 \sqrt{J} \sqrt{1 + \sqrt{1 + 6.76 \left( \frac{0.7(65) \cdot 294(26.625)}{29000 J} \right)^2}}$$

$$L_r = 61.7392 \sqrt{J} \sqrt{1 + \sqrt{1 + 6.76 \left( \frac{12.2815}{J^2} \right)}}$$

$$J = \sum \frac{1}{3} a b^3$$

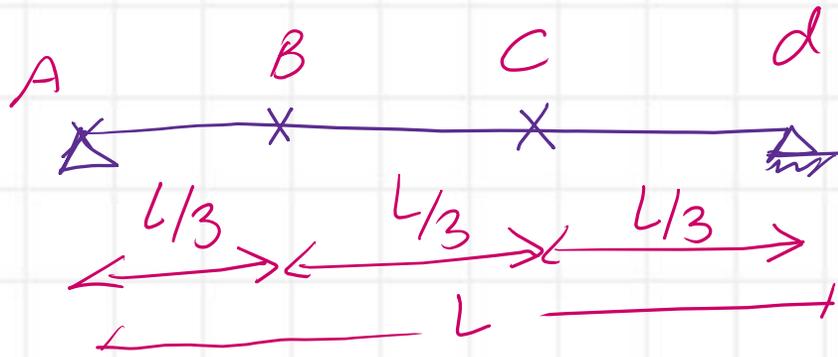


$$J = \frac{1}{3} \sum \left( (16) \left( \frac{5}{8} \right)^3 (2) + 26 \left( \frac{5}{16} \right)^3 \right)$$

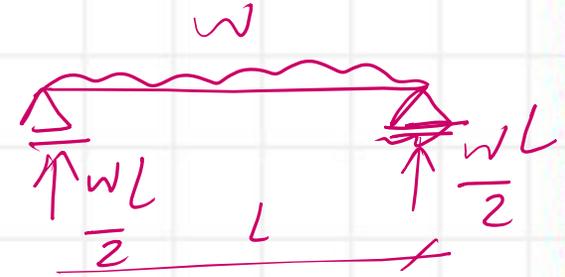
$$J = \frac{1}{3} [7.8125 + 0.79346] = 2.868 \text{ inch}^3$$

$$L_r = 61.7392 \sqrt{2.868} \sqrt{1 + \sqrt{1 + 123.962}}$$

$$L_r = 104.575 \sqrt{1 + 11.1786} = 104.575(3.4897) \rightarrow \div 12 \rightarrow = 30.41'$$

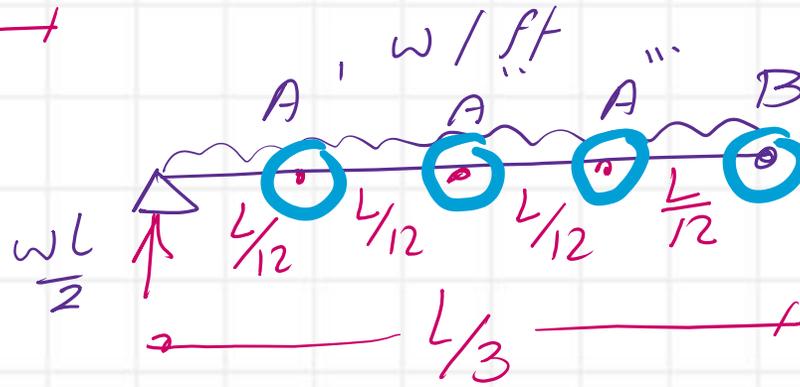


$$L = 4S'$$



For left part  
right part

$$\frac{A-B}{C-d}$$



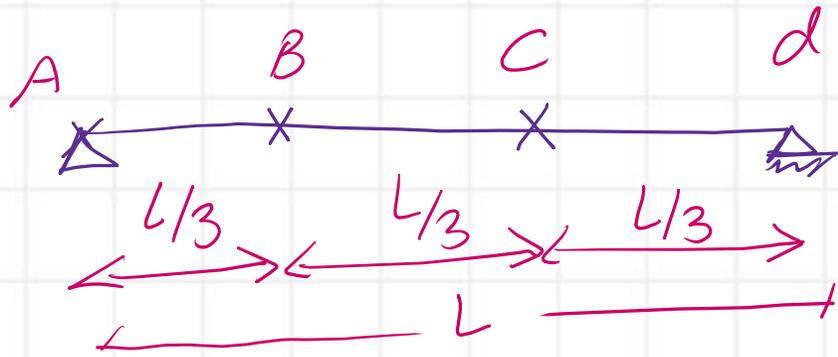
Divide  
to Four  
parts

Divide each part  
into 4 Quarters,  $A'$ ,  $A''$ ,  $A'''$

Find moment value at Each point

$$M_{A'} = \frac{wL}{2} \left( \frac{L}{12} \right) - w \left( \frac{L}{12} \right) \left( \frac{L}{24} \right) = \frac{wL^2}{12(24)} (11)$$

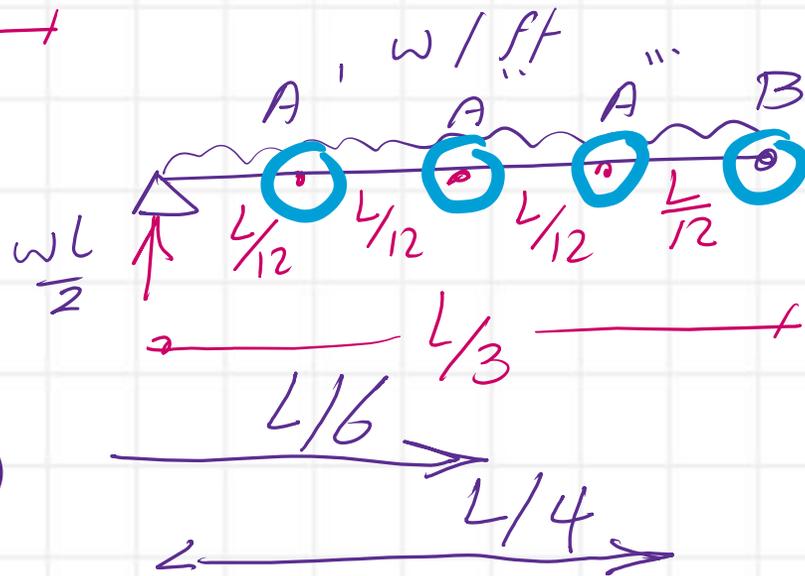
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$$L = 4s'$$



For left part  $\frac{A-B}{C-d}$   
right part



Divide to Four parts

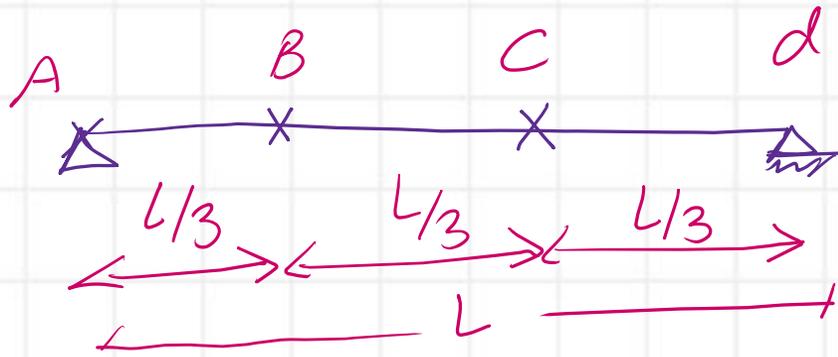
Find moment at  $A''$

$$\frac{WL}{2} \left( \frac{L}{6} \right) - w \left( \frac{L}{6} \right) \left( \frac{L}{12} \right)$$

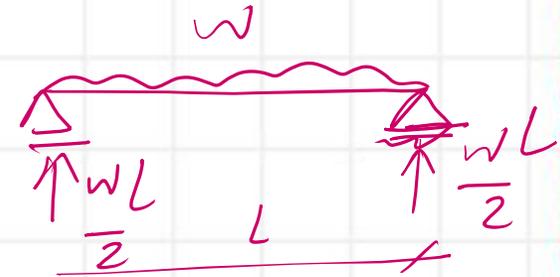
$$M_{A''} = \frac{WL^2}{72} (6 - 1) = \frac{5WL^2}{72}$$

$$M_{A'} = \frac{WL}{2} \left( \frac{L}{4} \right) - wL^2 \left( \frac{L}{4} \right) \left( \frac{L}{8} \right) = \frac{WL^2}{32} (4 - 1) = \frac{3WL^2}{32}$$

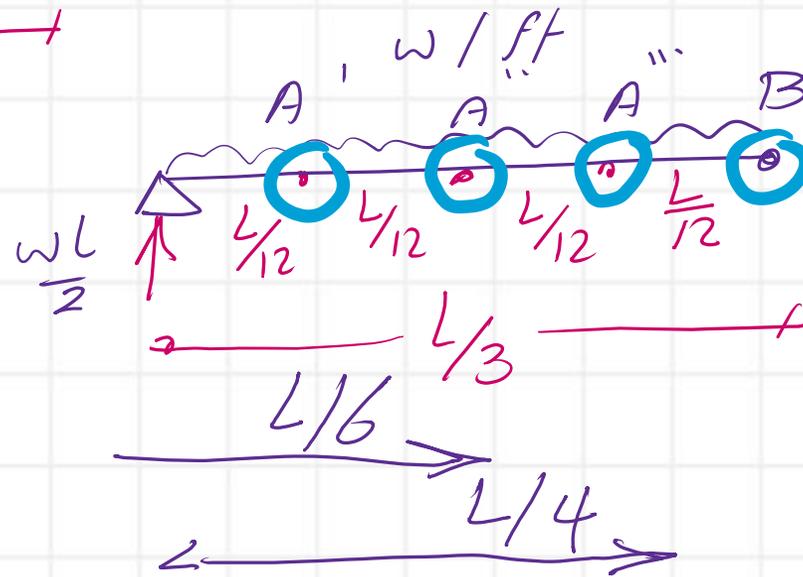
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$$L = 4s'$$



For left part  $A-B$   
right part  $C-d$



Divide to Four parts

Find moment at  $B$

$$\frac{wL}{2} \left( \frac{L}{3} \right) - w \left( \frac{L}{3} \right) \left( \frac{L}{6} \right)$$

$$M_B = \frac{wL^2}{18} (3 - 1) = wL^2 \left( \frac{2}{18} \right) = \frac{wL^2}{9}$$

$$C_b \text{ Value} = \frac{12.5 M_{max}}{2.5 M_{max} + 3 M_{A'} + 4 M_{A''} + 3 M_{A'''}}$$

$M_{max}$  max of  $C_b = 12.5 (wL^2/g)$

$$M_{A'} = \frac{11}{12(24)} wL^2$$

$$M_{A''} = \frac{5w}{72} L^2$$

$$M_{A'''} = \frac{3}{32} wL^2$$

$$M_B = \frac{wL^2}{9} \Rightarrow M_{max}$$

$$wL^2 \left( \frac{2.5}{9} + \frac{3(11)}{12(24)} + \frac{4(5)}{72} + 3 \left( \frac{3}{32} \right) \right)$$

$$C_b = (12.5/g)$$

$$\frac{80 + 33 + 80 + 81}{12(24)} =$$

$$C_b = \frac{(12.5/9)}{(80 + 33 + 80 + 81)} = \frac{12.5}{9} \frac{(12)(24)}{274}$$

$$C_b = 1.4598 \Rightarrow 1.46$$